Selection gui	ide of rotary	actuator												
SRC Step1	Osc	illating time che	eck											
Use (	Use oscillating time withing specified range of the below table													
HS Osci	Oscillating angle (°)													
R Mod	lel no.	90		180		270								
N	RRC-8		_	0.030 to 0.302	0.045 to 0.452									
H100	RRC-32			0.075 to 0.754	0.113 to 1.131									
AP * Osci	Illating time on table is	time to achieve the end of osc	illating	0.147 to 0.880	0.4	220 to 1.320								
SA2	induring time on table is		mannig	and starting movement.										
Step2	Size	e selection												
IA														
IAG If	clamp, or simp	le static forces, etc., a	are n	ecessary.	1	[]								
<p .<="" td=""><td>Static load</td><td></td><td></td><td></td><td></td><td></td><td></td></p>	Static load													
_A/B (1) W	Vorking pressure is de	etermined. P (MPa)		Calculation of required torque										
_AG/ BG (2) A	required force is det	ermined. F (N)	➡		➡									
EP (3) L	ength of an arm from		$T = F\ell (N \cdot m)$											
CP ac	tuator is determined.	ℓ (m)												
MF					-									
MFB • To	o move load		_		_									
-P	eistance load				]									
	/hon force /resister a	lood) coursed by fistional		Calculation of resistance										
GP for	rce gravity or other e													
1500 (1	) Working pressure	is determined. P (MPa)	→	$IR = K \times FR \times \ell (N \cdot M)$ K: clack coefficient										
3L (2	) A required force is		If load fluctuation free $K = 2$											
(3	) Length of an arm		If load fluctuates $K = 5$											
MD	actuator is determ	nined. (m)		(When resistance torque caused by		Determine size								
			]	gravity functions)		of rotary actuator								
	nertia load			if load fluctuates, when K < 5,		according to								
	To rotate body.			change of angular speed increases.		output torque								
(1) 0	scillating angle /oscillating tim	e and working pressure are determined.		₽	J	graph.								
	Oscillating angle	heta (rad)		+	1									
(9	Oscillating time	t (s)												
< <u>&lt;</u>	Working pressure	P (MPa)												
	90° = 1.5	708 (rad)		Requirea torque I = IR + IA	-									
	$180^{\circ} = 3.1^{\circ}$	410 (rad)												
<u>(2)</u>	∠≀∪ = 4./ Calculate load mom	ient of inertia according to		▲	J									
	oad shape and wei	ght. Refer to moment of			1									
	nertia table for the	calculation formula.												
	l (kg/m²)			Calculation of acceleration torque										
(3) A	Angular acceleration	n is calculated.	→	$  IA = 5 \times I \times \alpha (N \cdot m)$										
J	$\alpha = \frac{2\theta}{t^2}$ (rate	ad/s²)		I A IS THE REQUIRED TORQUE TO										
K	$\theta$ : Oscillat	ing angle (rad)		speed										
nding	t : Oscillat	ing time (s)												
						·J								
Step3	Che	eck of allowable	en	ergy										

When using an inertial load, keep the load energy to lower than the rotary actuator's allowable energy. (1) Calculate angular speed  $\omega = \frac{2\theta}{t}$  (rad/s)  $\theta$ : Oscillating angle (rad) t: Oscillating time (s) (2) Calculation of load inertia energy  $E=1/2l\omega^2(J)$ I: Load moment of inertia (kg/m<sup>2</sup>) Check if load inertia energy E to be allowable energy of rotary actuator or less. When exceeding allowable energy, external shock absorber, etc. is required.

## CKD

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## RRC Series

Selection guide

14													
ape					Radius of		GRC						
SNS	Sketch	Requirement	S	Moment of inertia I kg/m <sup>2</sup>	gyration K1 <sup>2</sup>	Remarks	RV3*						
0	$\mathbf{C}$	Diameter	d (m)	. Md²	d²	• No installation direction	NHS						
olate							HR						
	●Weight M (k	M (kg)	$(rg)$ $I = \frac{1}{8}$	8	When using with sliding, please consult with CKD	LN							
						ploade contrait with one	FH100						
dé	<u>~</u>	Diameter	$d_1$ (m) $d_2$ (m) section $M_1$ (kg) section $M_2$ (kg)	$I = \frac{1}{8} (M_1 d_1^2 + M_2 d_2^2)$	$\frac{d_1^2 + d_2^2}{8}$	●Ignore, when d₂ section is extremely small comparing to d₁ section	HAP						
h ste							BSA2						
Dial plate with	d						BHG						
		● Weight d₁ section d₂ section					LHA						
							LHAG						
				) $I = \frac{MR^2}{R^2}$	<u>R</u> <sup>2</sup>	The installation direction is horizontal     If vertical installation	HKP HLA/						
is an e	R		R(m)				HLB HLAG/						
otation		Bar length					HLBG						
	Weight	M (kg)	3	3	attitude, oscillating	HEP							
						time varies	HCP						
8							HMF						
$\subset$			$\begin{array}{c} R_1 \\ R_2 \\ M_1 \\ M_2 \end{array}$	$I = \frac{M_1 \cdot R_1^2}{3} + \frac{M_2 \cdot R_2^2}{3}$	$\frac{R_1^2 + R_2^2}{3}$	<ul> <li>The installation direction is horizontal</li> <li>If vertical installation attitude, oscillating time varies</li> </ul>	HIMEB						
rod	R2.	Bar length					пгр						
hin		<ul> <li>Weight</li> </ul>					HLC						
							HGP						
							FH500						
(ravity		● Bar length ● Weight	R(m) M (kg)	$I = \frac{MR^2}{12}$	<u>R<sup>2</sup></u> 12	No installation direction	HBL						
nter of g							HDL						
enter of rotation is cent	R						HMD						
							HJL						
							BHE						
)) (pa	'	<ul> <li>Plate length</li> <li>Length of side</li> <li>Weight</li> </ul>	a1 a2 b M1 M2	$I = \frac{M_1}{12} (4a_1^2 + b^2) + \frac{M_2}{12} (4a_2^2 + b^2)$	$\frac{(4a_1^2 + b^2) + (4a_2^2 + b^2)}{12}$	<ul> <li>The installation direction is horizontal</li> <li>If vertical installation attitude, oscillating time varies</li> </ul>	CKG						
lepip							CK						
gie pi							CKA						
lar pa							CKS						
angu							CKF						
(recta							CKL2						
eq	$\frown$	● Length of side ● Weight	a (m) b (m) M (kg)	$I = \frac{M}{12} (a^2 + b^2)$	$\frac{a^2 + b^2}{12}$	<ul> <li>No installation direction</li> <li>When using with sliding, please consult with CKD</li> </ul>	CKL2 CKL2						
elepip							-*-HC						
ctangular paralle													
							NCK/						
	a						SCK/FCK						
Ľ	$\rightarrow$						FJ						
							FK						
load		<ul> <li>Shape of concentrate</li> <li>Length to center of gradient</li> </ul>	ted load ravity of	$M_2 R_2^2$	Calculate k <sub>1</sub> <sup>2</sup> according to	<ul> <li>The installation direction is horizontal</li> <li>When M is avtramely</li> </ul>	Ending						
oncentrated I							/be						
	ncent			$I = M_1 (R_1^2 + k_1^2) + \frac{1012(N_2)}{3}$	shape of concentrated	small comparing to M1, may be calculated	/e t)						
	ů v v v v v v v v v v v v v v v v v v v						driv						
ပိ	Ârm M <sub>2</sub>		IVI2 (kg)		load	as $M_2 = 0$	tion						
low	to convert load J∟ to rotary actuator	shaft rotation when us	sing wit	h gear			ator						
Gear	b ConLoad IL	Gear Rotary side (the tooth	number) a	Moment of inertia of load retery shaft retation		Mhor share of	actu; on, I						
		Load side (the tooth number)		woment of inertia of load rotary shaft rotation		<ul> <li>vvnen snape of gear is increasing.</li> </ul>	ary a llati						
	JANE A IH	<ul> <li>Load inertia</li> <li>Moment</li> </ul>		$I_{H} = \left(\frac{a}{b}\right)^2 I_{L}$		gear moment of	<b>Rota</b> )sci						
	Rotary					inertia should be	шU						
	The second second	N⋅m											
	<u>u</u>						. –						
						CKD	15						